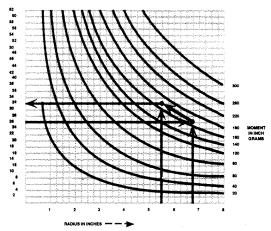
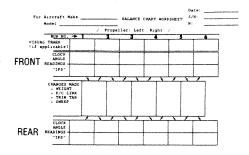
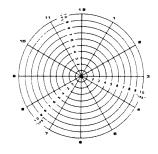
# **EXAMPLE TRIAL WEIGHT ADDITION AND CORRECTION**

When a good balance level is reached, determine the final weight installation point. By using the weight/moment graph, (below) find the moment of the existing trial weight by plotting at the intersection of weight and radius. Follow the curved moment line to the line that intersects the permanent radius. Read to the left to determine the weight requirement at the permanent location.



WEIGHT/MOMENT GRAPH

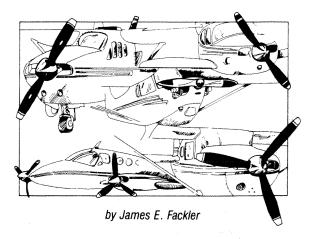


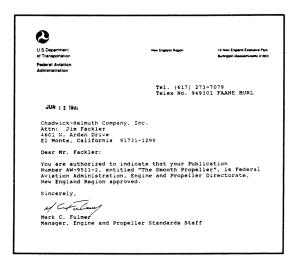




# The Smooth Propeller:

How to perform a dynamic propeller balance







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#### INTRODUCTION

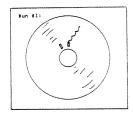
Propellers, spinners, crankshafts and other dynamic components are manufactured to strict tolerances. However, due to centrifugal force, aerodynamic loading and individual component tolerances some residual "out-of-balance" can be present. It has been determined that dynamic trim balance can correct such "out-of-balance" and results in significantly lower vibration levels.

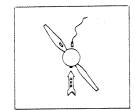
Because the vibration caused by an out of balance propeller is low to medium frequency, typically in the 15-45 Hz range, it is often felt throughout the airframe, usually at a lower amplitude than exists at the engine.

All propellers can benefit from dynamic balance. When properly executed the balance has both short and long term benefits. In the short term, there is usually an immediate reduction in the noise and

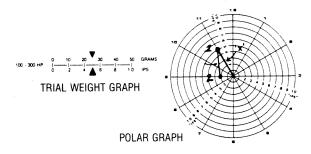
# EXAMPLE TRIAL WEIGHT ADDITION AND CORRECTION

Suppose an initial reading of 11:00 o'clock and 0.5 IPS. This point is plotted on the polar graph at the intersection of the IPS and angle and labeled point #1. From the trial weight graph (below) it can be determined the required weight will be approximately 25 grams.

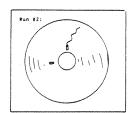


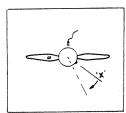


The vibration transducer is pointing up so the target is positioned where it was seen and the weight added opposite the sensor at 6:00 o'clock.



The second run yielded a reading of 9:00 o'clock and .21 IPS. This point is plotted on the polar graph and a line drawn from the first to second points, and from the first point to the center of the chart.





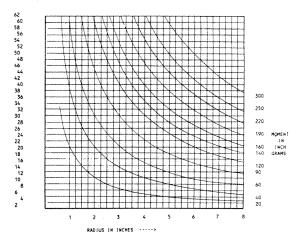
Position the propeller target where it was seen on the second run. Move the weight TOWARDS the original 6:00 o'clock position the same number of degrees it missed the center of the chart. More weight will also be required since the length of the line between the first and second points is shorter than the line between the first point and the center.

#### TABLE 2

#### WEIGHT/MOMENT CALCULATIONS

If the weight is moved inboard or out board from its trial location, it will be necessary to recompute the required weight based on the formula  $W \times A = M$ . We quals the weight. A equals the arm usually expressed as a station from the center of rotation. Mequals the moment, or the calculated force the weight produces, and is expressed as both a weight unit and distance measure (Example: 47 inch grams).

The following chart will aid in computing weight changes due to arm changes.

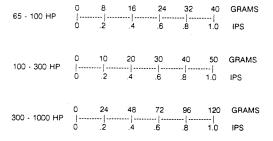


# TABLE 3 INITIAL WEIGHT CALCULATION

The initial weight calculation is dependent on the total weight of the propeller and the radius at which the weight is placed. If the weight of the propeller is known a trial weight roughly equivalent to 0.1% of the propeller weight applied to the spinner periphery would be adequate to produce enough change on which to base further calculations. In any case the record of the balance change as plotted on the polar coordinate chart is the best guide as to the effect of any weight.

Numerous other factors can alter the purely mathematic determination of the required weight such as mount stiffness, freedom of movement, transducer location, RPM, and sympathetic resonance to name only a few. This is why identical installations can result in different weight requirements for a given imbalance.

## DO NOT EXTRAPOLATE BEYOND GRAPH LIMITS



vibration "feel" in the cabin. Long term benefits are a reduction in the cycle fatigue of airboxes, exhaust stacks, radio equipment, sheet metal and cowling cracking and a reduction in wear of controls linking fixed portions of the airframe to the engine (i.e. prop control, carburetor, and mixture linkages).

The dynamic balancing process consists of mounting a vibration transducer (Velocimeter) to the engine and connecting the transducer by a cable to a Vibration Analyzer/Dynamic Balancer and Strobex (see photos below). The vibration transducer converts the motion produced by an out of balance condition to a weight and azimuth solution. Trial weights are then added to the propeller assembly as an experimental condition. These weights are fine tuned until the balance levels are acceptable, then a permanent installation is made.

This balance is generally good until the combination of components is upset via overhaul, or other major PTRC (Power Train Rotating Component) change.

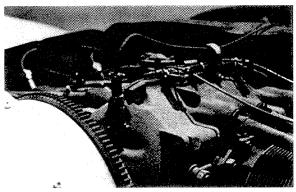
Dynamic trim balance relies on proper working condition of the PTRC. If deficiencies exist in the PTRC it is not possible to affect a dynamic balance. Such items as bent or broken crankshafts, bent blades, or mechanically faulty propeller assemblies are not balanceable by virtue of the physical dynamic properties of components with any freedom of migration. In short, dynamic balance uncovers faults, it does not mask or hide faults.

Other vibration sources may still be present after dynamic balance of the PTRC. Dynamic balance of the PTRC does not mask other vibration sources. These vibrations are in most cases produced by the engine as a result of combustion or engine accessories. Dynamic balance at the propeller RPM is not affected by these vibration sources since they occur at frequencies other than the PTRC. It should be understood that the vibration complexion of the aircraft will be improved after balance of the PTRC but in no way is the balance a panacea for the entire vibration character of the aircraft.

In summary, balance of the PTRC will not adversely affect any operation of the engine or related components. Impending failures due to other sources of vibration such as firing order, detuned counterweights, and out of balance or misaligned components will continue to take their toll. Since failure due to vibration is additive, any reduction will relieve stress to all the affected parts. Parts will continue to break, but at reduced intervals.



Balance reading being taken with a Vibration Analyzer/Dynamic Balancer and Strobex.



Typical vibration transducer (Velocimeter) installation on engine

#### PRE BALANCE INSPECTIONS

Ensure that all Airworthiness Directives are complied with for the particular propeller in question.

Inspect the blades for nicks, scratches and gouges. Dress out as specified per the appropriate propeller manufacturer.

Painting, boot replacement, and dressing should be done prior to dynamic balance.

Perform a visual inspection of the propeller and hub assembly checking for security, and proper installation.

Inspect the spinner and spinner bulkhead for cracks, stopdrills, and weldments. Dynamic balance weights may not be applied to spinners with any of these conditions present.

Remove any existing dynamic balance weights. Record the number and position of static weights if so equipped.

## **EQUIPMENT CALIBRATION**

Ensure that the balance equipment has been function checked with equipment traceable to National Bureau of Standards within the preceding 6 months.

The Chadwick-Helmuth Calibrator Model 11, 11A, or 11/11PS is acceptable for this function test. See manufacturers instructions for calibration procedures.

### INSTALLATION OF EQUIPMENT

Vibration transducer attachment

Attach vibration transducer (Velocimeter) within 4 inches of the power output end of the engine crankcase so its long axis is vertical and its connector is up. Alternately the connector may point down, the trial weight installation will change per instructions found in TRIAL WEIGHT ADDITION section. On most engines the forward most case half bolt is an excellent location.

### TROUBLESHOOTING GUIDE

The following chart may be used to isolate problems in balance.

# I EQUIPMENT MALFUNCTION

- A) Transducers/Cables
  - 1) Vibration transducer (Velocimeter)
  - 2) Cable open or short
- B) Balancer
  - 1) Power supply
  - 2) Filter (no IPS)
  - 3) Mechanical damage (switches, knobs, etc.)
- C) Strobex
  - 1) Power supply
  - 2) Flash tube or flash tube focus
  - 3) Will not trigger due to lack of trigger pulse

#### II OPERATOR ERROR

- A) Balancer/Strobex controls incorrectly set
- B) Not tuning properly or tuned to wrong RPM
- C) Error in reading IPS or clock angle

### III AIRCRAFT/INSTALLATION PROBLEM

- A) Poor vibration transducer location
- B) Faulty PTRC components
- C) Loose cable or vibration transducer mounts

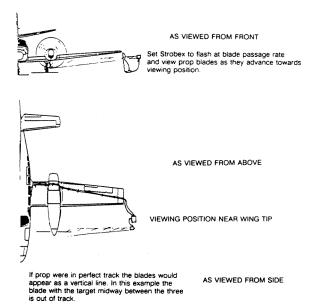
#### TABLE

### HARDWARE REQUIREMENTS

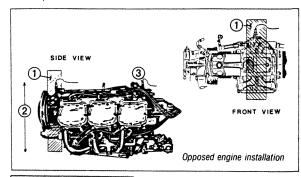
- 1. All balance hardware is to be aircraft grade or equivalent.
- 2. All hardware to be corrosion compatible per AC 43.13. Stainless hardware is preferred, cadmium plating is acceptable as a corrosion inhibitor on ferrous metals.
- Any lead weights used must be supported between steel washers having at least 80% surface contact with the lead.
- All nuts used to secure balance weights must be of the self locking type.
- Minimum acceptable bolt diameter is 3/16 inch. Maximum bolt diameter not to exceed 1/4 inch.
- All holes in the bulkhead are to be drilled undersize and reamed to final dimensions.

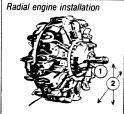
# APPROXIMATE WEIGHT OF AN HARDWARE

BOLTS	WASHERS	NUTS	GRAMS
		MS2104-103	32} 1.0
	AN 416	AN364-1032	
AN3-3A		AN365-1032	3.0
		AN365-428	-
AN3-5A			+ 4.0
	AN970-3		5.0
AN3-7A			나 6.0
AN4-3A			
AN4-5A			다 7.0 -
A114.0A			8.0
AN4-6A			7
AN4-10A	AN970-4 -		9.0
7477 1071			10.0
AN4-12A			



On radial engines, the preferred installation is horizontal slightly aft of the output shaft.





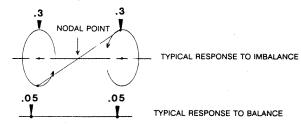
#### CCEND

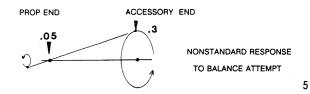
- Area in which vibration transducer should be located
- 2. Propeller plane of rotation
- Nodal survey vibration transducer (Velocimeter)

# INSTALLATION OF EQUIPMENT

Usually when a propeller is out of balance, it causes the entire engine to oscillate about it's C.G. (Center of Gravity). As the heavy point of the propeller is going up, the rear of the engine is going down. During a routine balance, the amplitude (amount) of vibration produced by the propeller orbit is about the same at the front and rear of the engine, and as the balance is improved both the front and rear of the engine decrease in amplitude AND THE ENGINE FEELS NOTICEABLY BETTER.

If there is a problem elsewhere in the engine or there is a problem in the propeller assembly, it will be evident as the rear vibration transducer ③ reading will remain the same or increase in intensity. If this is the case, the propeller track should be checked as outlined under propeller track. The nature of the problem should be determined before attempting to balance.





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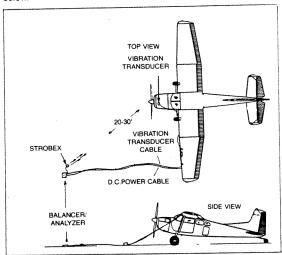
# TARGET ATTACHMENT AND CABLE ROUTING

1. Outside viewing

Attach a reflective target P/N 3300 to the root of a single blade. The target will be observed from the front of the aircraft.

Position the cables clear of moving parts to a convenient viewing position ahead and slightly off center line, 20 to 30 feet away from the aircraft.

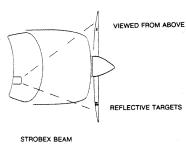
Attach the cables to the Balancer/Analyzer and Strobex as illustrated below.

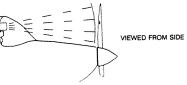


2. Inside viewing

Attach a reflective target P/N 3300 to the face of a blade where it will be visible from inside the aircraft. Attach two targets on another blade if it is a two bladed prop, and three targets on the third blade of a three bladed prop, etc.

Because the visible arc of the propeller is limited when viewed from inside it is necessary to mark all the blades so angle of the master blade (#1) can always be determined. Each blade is marked individually so there is no confusion in identity.





### PROPELLER TRACKING

Installation of a vibration transducer is not required for blade tracking.

Attach a reflective tape overlapping the leading edge of a blade approximately 1 inch from the tip. Attach another reflective tape to the next blade in rotation overlapping the leading edge approximately 2 inches from the tip. Continue application to additional blades moving the target position a little farther in on each subsequent blade. (See illustrations, below).

Operate aircraft at idle. Using the Strobex, adjust the oscillator to fire at the rate at which blades pass. (Example: 3 blades at 1000 RPM = 3000 blade pass rate).

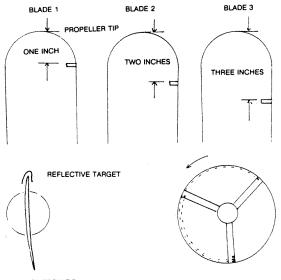
Observe the targets on the blade tips edge on to the propeller disc. Fine tune the oscillator until the targets come into view. The fore and aft relationship of the targets will be equal with a perfect track. If a target is ahead or behind the other target(s) the prop is out of track in the obvious sense. (See illustrations, p. 16).

Operate the aircraft at a medium power setting. Adjust the oscillator to the new blade pass rate. Fine tune to stop the target image. Observe and record the track relationship.

Repeat readings at a higher power setting.

Differences of 3/16 inch or more are likely to create problems when attempting to balance. If the fore and aft relationship "floats" or changes during any of the settings there is likely a wear problem in the propeller hub or blade retention mechanism. If the track picture changes between power settings it may indicate a "soft" blade or a blade whose lift characteristics change due to profile, incidence, or bending. In any of the above cases it is recommended that the blade thickness, chord width and diameter be checked against propeller manufacturers limits and corrected if required. Differences that are constant over all RPMs are usually caused by blade angle, microadjusting ring, or hub shimming differences.

Some looseness in the hub is noticeable in the static condition. This is a normal condition on most propellers and is acceptable. See propeller manufacturers limits for excessive amounts.



AS VIEWED FROM TIP TOWARDS ROOT OF BLADE

15

### DOCUMENTATION/PLACARDS

The propeller hub must bear a placard notifying future maintenance personnel the propeller has been dynamically balanced and the assembly of the PTRC is an indexed assembly.

The maintenance entry should be made in the appropriate propeller and/or engine log book and should reference the amount of weight installed and its azimuth. Azimuth reference can be made to either top dead center or a clearly visible indelible mark on the PTRC. This indelible mark must be made on all accessible PTRC components.

Enter the engine time and date and the final IPS and azimuth readings.

The entry must be signed and the certificate number of the maintenance person entered.

If the balance was done using a hard copy vibration recorder include a copy of the before and after vibration record.

In normal service the balance will last approximately 400 hours of average use, on conventional geared airplanes (tail draggers) approximately 700 hours. Normal preventative maintenance of the leading edges on metal propellers will not normally alter the balance enough to require a rebalance. Nicks need to be removed in accordance with the manufacturers guidelines. Advisory Circular 20-37D outlines the care of metal propellers including dressing of nicks, scratches, and gouges. After dressing of the blades the balance must be rechecked to confirm the balanced condition has not been disturbed.

#### LIMITATIONS

No propeller may be balanced whose initial reading exceeds 1.2 inches per second (IPS) velocity.

NOTE: If the unbalanced reading exceeds 1.2 IPS it is recommended the propeller be removed and re-static balanced.

No weights may be attached to spinners or spinner bulkheads which are cracked, stopdrilled or welded.

Balance washers may be applied to the outside of the spinner for test purposes only. Washers on the outside of the spinner are not permitted as a permanent installation.

Self adhesive weights are not permitted except where outlined by the propeller manufacturer.

Acceptable balance level is .2 inches per second (IPS) velocity at the RPM of the propeller.

Clearance of attachment hardware and weight must be a minimum of .25 inch from fixed portions of the cowling and engine, except the engine where outlined for starter ring gear attachments.

### BALANCE/VIBRATION CHECK

#### CALITION

Observe engine operating limits. Airflow to the engine is generally limited during ground operation and may result in reaching temperature limits rapidly.

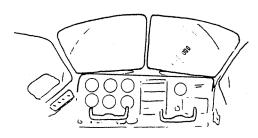
The balance reading may be taken at any RPM. Low cruise RPM on most opposed engines up to 300 HP is a good ballpark figure.

Observe propeller in arc circumscribed by reflective target(s) using Strobex. Observe the angle of the target and the amount of vibration as shown on the balancer/analyzer and record.

WARNING – If IPS (Inches per Second Velocity) exceeds 1.2 at propeller RPM do not attempt balance. Remove propeller and re-static balance assembly or re-phase propeller on crankshaft if permitted by manufacturer.

For complete instructions on equipment used consult the particular operating manual for that equipment.

In the illustration below the third of three blades appears at 1:00 o'clock. Normally the first target seen is used as a reference or master blade whose position is plotted on the polar coordinate chart. The IPS level is displayed on the balancer/analyzer.



### TRIAL WEIGHT ADDITION

- 1. With the engine shut down, rotate the propeller to the angular position at which it was observed while running.
- 2. Place a trial weight on the propeller, OPPOSITE THE DIRECTION THE VIBRATION TRANSDUCER POINTS on the engine. See examples on page 8.

# WARNING/WARNING/WARNING

Trial weights may be added to the spinner retention screws using large area washers (with longer screws if necessary) for purposes of test. These are TRIAL weights only and are prohibited as a permanent installation. Do not exceed a stack of six washers on any screw.

# CAUTION/CAUTION/CAUTION

Trial balance weights may not be added to the spinner retention screws if they are smaller than #10 (10-32) size.

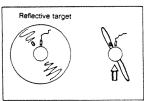
3. How much weight to use depends on the propeller, but as a rule the following should provide adequate guidance for a first run.

Rated takeoff horsepower/Grams per .1 IPS at spinner radius 100-300/5.0-6.0 300-800/10.0-12.0

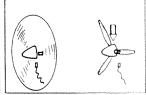
800 + /Consult Chadwick-Helmuth Co.

#### TRIAL WEIGHT ADDITION EXAMPLES

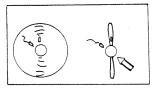
- 1. When engine is shut down, rotate propeller target blade to angle at which it was observed while running.
- 2. Place weight on propeller OPPOSITE DIRECTION VIBRATION TRANSDUCER POINTS on the engine.



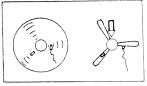
Vibration transducer pointing up. Add weight at 6:00 o'clock



Vibration transducer pointing down. Add at 12:00 o'clock position



Vibration transducer pointing toward 10:30. Add weight at 4:30



Vibration transducer pointing down. Add at 12:00 o'clock position

### PLOTTING BALANCE POINTS

1. Run the engine again at the same power setting and repeat the readings.

Second reading below .2 IPS:

2. If on the second run the reading was below .2 IPS proceed to permanent installation instructions.

Second reading above .2 IPS:

Using balance chart worksheet #3659 on p. 20 plot the first and second points at their respective intersections of angle and IPS. Label them point #1 and point #2.

On the target clock (page 9) connect the two points with an arrow so the arrow points at the second point.

Draw a line between the first point and the center of the chart.

If the arrow is longer than the line, proportionally less weight is required.

If the arrow is shorter than the line, proportionally more weight is required.

Measure the angle between the line and the arrow.

If the target was viewed in a clockwise direction from the first point the weight must be moved the angular amount counterclockwise on the propeller.

#### STARTER RING GEAR

#### WARNING/WARNING/WARNING

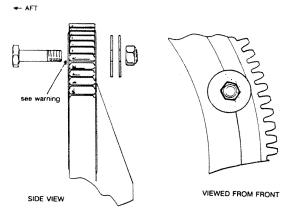
DO NOT install washer under head of bolt. Head of bolt must face engine side of ring gear.

Hardware used: \*AN4-"X"A bolt AN970-4 washer AN365-428 nut

\*On some ring gears the weight attach holes are countersunk on the engine side of the ring gear. These ring gears necessitate the use of countersunk machine bolts on the engine side of the ring gear in lieu of the AN 4 bolt.

A maximum of two holes may be used. Bolt head must have positive clearance through starter bendix with crankshaft seated fully aft. This method is to be used only on ring gears without spinner bulkheads attached.

Note: Attachment hardware and weight must clear stationary portions of the cowling by at least .25 inch.



# STATIC BALANCE WEIGHT LOCATIONS

Use of the static weights is limited by some propeller manufacturers. Consult propeller overhaul instructions for applicable part numbers and number of weights that can be installed.

If the static weights are used for dynamic balance the P/N (or equivalent gram weight) and location of these weights must be recorded in the logbook so the propeller can be restored to its original static balance condition if placed on another aircraft.

It will be necessary to recompute the balance weight from the outside of the spinner to the static weight locations. This is essentially a weight and balance recomputation based on the formula  $W \times A = moment$ , where W = weight, A = arm (radius from center of rotation).

# INSIDE PERIPHERY OF SPINNER BULKHEAD

Note: If required weight exceeds 200 inch grams in a single location divide weight equally between two holes. A maximum of two holes may be used.

Hardware requirement: MS 24694-S"X" countersunk machine screw

AN 970-3 washers MS 21042-L3 lock nut

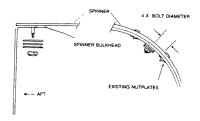
AN 365-1032A lock nut (alternate)

Installation note:

First washer against bulkhead may require trimming to accommodate radius of bulkhead flange.

First washer against bulkhead may require light countersinking to provide proper torque values. No more than two holes allowed to fit balance weights.

Type 3 Countersunk machine screw, weights inside bulkhead



#### SPINNER BACKING PLATE

Type 4 Axial bolt, doubler washer

Minimum thickness: Maximum radius: .040 inch .0 3 inches

.040 inch 5 inches 2850 RPM

Maximum rotational velocity: Maximum moment:

2850 RPM 2850 RPM 180 inch grams 180 inch grams

Note: If weight required exceeds 90 inch grams in a single location, divide equally between two holes. A maximum of two holes may be used.

Note: Attachment hardware and weight must clear stationary portions of the cowling and engine by at least .25 inch.

Hardware requirement: AN 3-"X"A

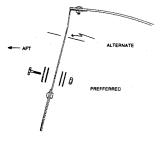
3-"X"A

AN 970-3 washer AN 960-316 washer

AN 365-1032A nut

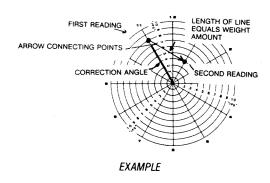
Note: Preferred location near reinforcing web. Alternate to be used to meet cowling clearance requirements. Washers must be placed on both sides of the bulkhead.

Type 4 Axial bolt, doubler washer



### PLOTTING BALANCE POINTS

If the target was viewed in a counterclockwise direction from the first reading the weight must be moved the angular distance clockwise on the propeller.



#### **VERIFICATION RUN**

Run the aircraft again at the RPM of the first two readings. Continue to adjust the balance weights until the IPS level is less than 0.20.

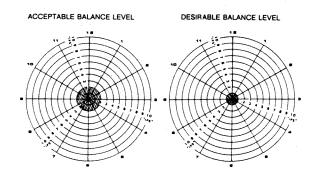
The weight should not be spread over more than 3 retention screws, or farther than 40 degrees of azimuth unless it is necessary due to the angular spacing of the spinner screws.

#### CAUTION/CAUTION/CAUTION

Trial weights must not be added to spinner screws smaller than #10 (10-32) size.

Once the balance is reduced to acceptable levels, record final IPS and Clocking information for inclusion in maintenance records.

If the balance cannot be reduced to acceptable levels within 4-5 runs and/or the readings are not repeatable or predictable it is evidence there is a problem with the PTRC that is making balance unlikely. Loose or worn components, out of track, etc., are often the cause. Balance weights may not be applied to the PTRC that exhibits these readings. Proceed to section on propeller tracking p. 15 and section on troubleshooting p. 17.



#### **GENERAL PRECAUTIONARY NOTES**

Be absolutely certain the balance weights do not interfere with the operation of the propeller. Propeller cuffs, counterbalances, deice assemblies, etc. must all operate in a normal manner with balance weights installed.

Rotate the propeller through by hand after weight addition. Look as well as listen for clearance conflicts. Bear in mind as the propeller produces thrust it will force the propeller and spinner forward. Also remember the transient vibration on startup and shut down will require more than just positive clearance.

DO NOT USE STICK ON SELF ADHESIVE WEIGHTS. This is prohibited unless specified by the propeller manufacturer.

Do not file or paint the blades to affect balance. Filing of the blades has an aerodynamic as well as mass balance effect.

#### WARNING/WARNING

Before turning any propeller, ensure the magneto switches are off and in proper working condition (i.e. grounded mags).

### WEIGHT INSTALLATIONS

There are four basic acceptable methods of weight attachment to the PTRC. Variations of each addition method are listed as subtypes

Note: Reference to axial and radial are from the crankshaft longitudinal axis. Axial is parallel to the crankshaft long axis, radial is 90 degrees to the crankshaft centerline.

- 1. Inside periphery of spinner bulkhead
  - A) Type 1 Radial bolt, doubler washer
  - B) Type 2 Countersunk machine screw, aft facing flange
  - C) Type 3 Countersunk machine screw, forward facing flange
- 2. Spinner backing plate
  - A) Type 4 Axial bolt, doubler washer
- 3. Starter ring gear
- 4. Static balance weight locations

# **INSIDE PERIPHERY OF SPINNER BULKHEAD**

Type 1 Radial bolt, doubler washer

Minimum bulkhead thickness: .030 inch Maximum radius: 6.75 inch 2850 RPM Maximum rotational velocity: Maximum moment: 350 inch grams

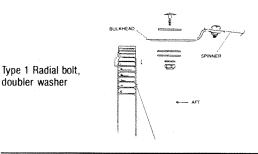
Hardware requirement: AN 970-3 washer AN 960-316 washer nut

AN 365-1032A AN 525-10R"X" screw

Note: A maximum of two holes may be used for attachment of balance

Note: Weight and attachment hardware must clear stationary portions of the cowling and engine by at least .25 inch.

# INSIDE PERIPHERY OF SPINNER BULKHEAD



Type 2 Countersunk machine screw, aft facing flange

Minimum bulkhead thickness: .040 inch Maximum radius: 6.75 inch Maximum rotational velocity: 2850 RPM 400 inch grams Maximum moment:

Note: Divide weight equally between two holes if moment exceeds 200 inch grams in a single location. A maximum of two holes may be used.

Hardware requirement: MS 24694-S"X" countersunk machine screw

AN 970-3 washers MS 21042-L3 lock nut AN 365-1032A lock nut (alternate)

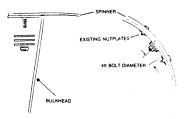
Note: Weight and attachment hardware must clear stationary portions of the cowling and engine by at least .25 inch.

doubler washer

First washer against bulkhead may require trimming to accommodate radius of bulkhead flange.

First washer against bulkhead may require light countersinking to provide proper torque values. No more than two holes allowed to carry balance weights.





Type 3 Countersunk machine screw, weights inside bulkhead

Minimum bulkhead thickness: .040 inch Maximum radius: 6.75 inch Maximum rotational velocity: 2850 RPM Maximum moment: 400 inch grams